

# (amputation of $H_\infty$ Norm for Flexible Structures

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**Abstract.** The  $H_\infty$  norm for a system with flexible structural properties is a function of its largest Hankel singular value.

The computation of the  $H_\infty$  norm of a linear system is a computationally intensive search process [1-3], specifically in the case of flexible systems. This paper presents a direct way to determine the  $H_\infty$  norm for flexible structures which avoids iterations.

In this paper a flexible structure is defined as a controllable and observable linear system with distinct complex conjugate pairs of poles ( $N$  poles,  $N$  is even), and with small negative real parts of the poles. In the Moore balanced coordinates [4] it consists of  $n = N/2$  components [5, 6], and each component consists of two states. Let  $(A, B, C)$  be a state-space triple of a flexible structure, and  $G = C(sI - A)^{-1}B$  its transfer function, with the  $H_\infty$  norm defined as follows

$$\|G\|_\infty = \sup_{\omega} \sigma_{\max}(G(j\omega)) \quad (1)$$

The system controllability and observability grammians  $W_c$  and  $W_o$  are positive-definite and satisfy the Lyapunov equations

$$AW_c + W_cA^T + BB^T = 0, \quad A^TW_o + W_oA + CC^T = 0. \quad (2)$$

The system representation is balanced in the sense of Moore (cf. [4]) if its controllability and observability grammians are equal and diagonal

$$W_c = W_o = \Gamma^2, \quad \Gamma = \text{diag}(\gamma_1, \dots, \gamma_n) \quad (3)$$

where  $\gamma_i > 0$  is the  $i$ -th Hankel singular value of the system, and the Hankel singular values are in the decreasing order,  $\gamma_i \geq \gamma_{i+1}$ ,  $i = 1, \dots, n-1$ .

Further, an approximate equality between two variables is used in the following sense. Two variables  $x$  and  $y$  are approximately equal ( $x \approx y$ ) if  $x = y + \epsilon$ , and  $\|\epsilon\|/\|y\| \ll 1$ . It is shown, see [5, 6], that for a balanced flexible system  $(A, B, C)$  with  $n$  components (or  $N = 2n$  states), the balanced grammian has the following form

$$\Gamma = \text{diag}(\gamma_1, \gamma_1, \gamma_2, \gamma_2, \dots, \gamma_n, \gamma_n) \quad (4)$$

and the matrix  $A$  is almost block-diagonal, with dominant  $2 \times 2$  blocks on the main diagonal

$$A \approx \text{diag}(A_i), \quad A_i = \begin{bmatrix} -\zeta_i \omega_i & -\omega_i^2 \\ \omega_i & -\zeta_i \omega_i \end{bmatrix}, \quad i = 1, \dots, n \quad (5)$$

where  $\omega_i$  is the  $i$ -th natural frequency of the structure, and  $\zeta_i$  is the  $i$ -th modal damping. Note also that introducing (4) and (5) to (2) gives

$$\gamma_i^2(A_i + A_i^T) \approx B_i B_i^T \approx -C_i^T C_i \quad (6)$$

and  $B_i, C_i$  are  $i$ -th row and column of  $B$  and  $C$ , respectively.

The  $H_\infty$  norm is evaluated from the Riccati equation, as shown in [1-3]. It is the smallest positive parameter  $\rho$  such that the solution of the following Riccati equation is positive-definite

$$A^T S + S A + \rho^2 S B B^T S + C^T C = 0. \quad (7)$$

From this definition it follows that:

**Proposition.** For  $G$  being a transfer function of a flexible structure in the state space representation  $(A, B, C)$ , its  $H_\infty$  norm is as follows

$$\|G\|_\infty = 2\gamma_1 \quad (8)$$

where  $\gamma_1$  is the largest Hankel singular value of the system.

**Proof.** For a flexible structure, due to properties (4)-(6) the solution  $S$  of the Riccati equation (7) is diagonally dominant, see [7]

$$S \approx \text{diag}(s_1, s_1, \dots, s_n, s_n) \quad (9)$$

where  $s_i$  is a solution of the following equation

$$s_i(A_i + A_i^T) + s_i^2 \rho_i^2 B_i B_i^T + C_i^T C_i = 0, \quad i = 1, \dots, n \quad (10)$$

and  $A_i, B_i$ , and  $C_i$  are given in (5). Introducing (6) to (10) one obtains

$$s_i^2(A_i + A_i^T) - s_i \rho_i^2(A_i + A_i^T) - \gamma_i^2(A_i + A_i^T) = 0 \quad (11)$$

or

$$s_i^2 - s_i \rho_i^2 \gamma_i^{-2} - \rho_i^2 = 0 \quad (12)$$

with two solutions  $s_i^{(1)}$  and  $s_i^{(2)}$

$$s_i^{(1)} = 0.5 \rho_i^2 \gamma_i^{-2} (1 - \beta_i), \quad s_i^{(2)} = 0.5 \rho_i^2 \gamma_i^{-2} (1 + \beta_i), \quad (13a)$$

$$\beta_i = \sqrt{1 - 4 \gamma_i^4 / \rho_i^2} \quad (13b)$$

For  $\rho_i = 2\gamma_i^2$  one obtains  $s_i^{(1)} = s_i^{(2)} = 2$ , and  $\rho_i = 2\gamma_i^2$  is the smallest value of  $\rho$ , for which a positive solution  $s_i$  exists. It is indicated by the plots of  $s_i^{(1)}$  and  $s_i^{(2)}$  vs  $\rho$ , in Fig. 1. To obtain  $S$  positive definite, all  $s_i$  must be positive. Thus the largest  $\rho$  from the set  $\{\rho_1, \rho_2, \dots, \rho_n\}$  is the smallest one for which  $S > 0$ , therefore

$$\|G\|_\infty = \max_{\rho_i} \rho_i = 2\gamma_1 \quad (14)$$

The proposition shows that the computation of the  $H_\infty$  norm consists of a standard procedure of computing the system Hankel singular values, and that the largest Hankel singular value determines the required norm.

*Example.* A truss structure as in Fig.2 is investigated. For this structure  $l_1 = 70$  in.,  $l_2 = 100$  in.; each truss has a cross-section area of  $22 \text{ in.}^2$ , elastic modulus of  $10^6 \text{ lb/in.}^2$ , and mass density of  $2 \text{ lb sec}^2/\text{in.}^2$ . Vertical control forces are applied at nodes *na1* and *na2*, and the output rates are measured in the vertical direction at nodes *no1* and *no2*. The system has 26 states (13 balanced components), two inputs, and two outputs. Its  $H_\infty$  norm is computed through iterations of Riccati equation (7), obtaining  $\|G\| = 1.340852$ . It took 24 iterations to obtain the required accuracy  $\epsilon = 10^6$ , as shown in Fig.3. The  $H_\infty$  norm is also obtained from (8), and in this case  $\|G\|_\infty = 1.340850$ , with the same accuracy  $\epsilon = 10^6$ .

In conclusion, the estimate of the  $H_\infty$  norm (8) is obtained without costly searching, with the estimation error close to the machine zero.

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#### References

1. J. C. Doyle, B.A. Francis, and A.R. Tannenbaum: *Feedback Control Theory*, Macmillan, New York, 1992.
2. S. P. Boyd, and C. H. Barratt: *Linear Controller Design*, Prentice Hall, Englewood Cliffs, NJ, 1991.
3. J. C. Doyle, P. P. Khargonekar, and B.A. Francis: "State Space Solutions to Standard  $H_2$  and  $H_\infty$  Control Problems," *IEEE Trans. Autom. Control*, vol. 34, No. 8, 1989, pp. 831-847.
4. B.C. Moore: "Principal Component Analysis in Linear Systems, Controllability, Observability and Model Reduction," *IEEE Trans. Autom. Control*, vol. 26, No. 1, Jan. 1981.
5. W. Gawronski, and J.N. Juang: "Model Reduction for Flexible Structures," in: *Control and Dynamics Systems*, ed. C.T. Tondes, vol. 36, pp. 143-222, Academic Press, New York, 1990.
6. W. Gawronski, and T. Williams: "Model Reduction for Flexible Space Structures," *Journal of Guidance, Control, and Dynamics*, vol. 14, No. 1, Jan, 1991, pp. 68-76.
7. W. Gawronski: "Balanced JQG Compensator for Flexible Structures," *1993 American Control Conference*, San Francisco, CA, 1993.

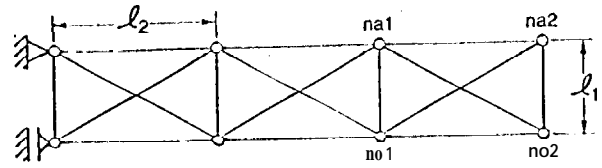


Fig.2. Truss structure

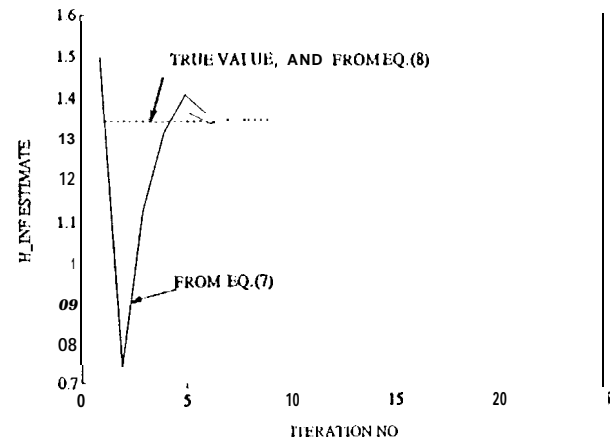


Fig.3.  $H_\infty$  norm of truss structure

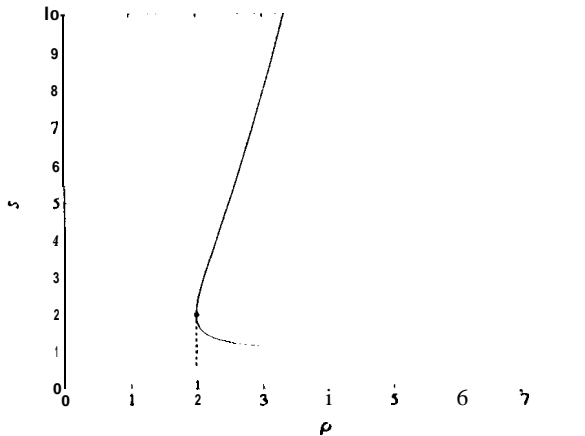


Fig.1. Solution of Riccati equation